

DYNAMICS OF STRUCTURES

Single Degree of Freedom System: Forced Harmonic Vibration (Cont..)

Resonance Response: Undamped System

- The harmonic vibration response of undamped system is given by:

$$u(t) = u_0 \cos \omega t + \frac{v_0}{\omega} \sin \omega t + \frac{p_0}{k} \frac{1}{1 - \beta^2} (\sin \Omega t - \beta \sin \omega t)$$

- For a simple case of $u_0 = v_0 = 0$

$$u(t) = \frac{p_0}{k} \frac{1}{1 - \beta^2} (\sin \Omega t - \beta \sin \omega t) = \frac{p_0}{k} \left(\frac{\sin \beta \omega t - \beta \sin \omega t}{1 - \beta^2} \right)$$

- For $\beta = 1.0$, the numerator and the denominator are both zero and the displacement becomes indeterminate. In the limiting case, the problem can be solved by *L'Hospital's rule*

$$\lim_{\beta \rightarrow 1} u(t) = \frac{p_0}{k} \lim_{\beta \rightarrow 1} \left(\frac{\sin \beta \omega t - \beta \sin \omega t}{1 - \beta^2} \right) = \frac{p_0}{k} \lim_{\beta \rightarrow 1} \left(\frac{\omega t \cos \beta \omega t - \sin \omega t}{-2\beta} \right)$$

$$u(t) = \frac{p_0}{k} \left(\frac{\omega t \cos \omega t - \sin \omega t}{-2} \right) = \frac{p_0}{2k} (\sin \omega t - \omega t \cos \omega t)$$

Resonance Response: Undamped System (Cont..)

- The response is periodic with period of $T = 2\pi/\omega$.

- Peak response is obtained by differentiating:

$$u = \frac{p_0}{2k} (\sin \omega t - \omega t \cos \omega t)$$

$$\dot{u} = \frac{p_0}{2k} (\omega \cos \omega t - (\omega \cos \omega t + \omega t (-\omega \sin \omega t)))$$

$$\dot{u} = \frac{p_0}{2k} \omega^2 t \sin \omega t$$

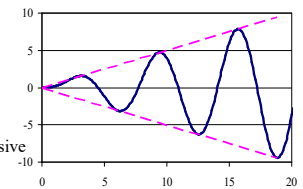
$$\dot{u} = 0 \Rightarrow \omega t = n\pi \quad \dots \quad n = 0, 1, 2, 3, \dots$$

- The difference between two successive peaks is given by

$$u \left(\frac{2\pi}{\omega} + \frac{n\pi}{\omega} \right) - u \left(\frac{n\pi}{\omega} \right) = \frac{p_0}{2k} 2\pi \cos n\pi = \pm \frac{p_0}{k} \pi$$

$$\frac{u_k}{p_0} = \frac{1}{2} (\sin \omega t - \omega t \cos \omega t)$$

Modification of last equation



Cos and sine is same for both t values. Only t value with Omega causes difference.

Damped Harmonic Vibration

- Damped Harmonic Vibration is governed by:

$$m\ddot{u} + c\dot{u} + ku = p_o \sin \Omega t \text{-----(1)}$$

- The particular solution (steady state response) is of the form:

$$u = G_1 \cos \Omega t + G_2 \sin \Omega t \text{-----(2)}$$

- On putting in eq.1 we get values of G_1 and G_2 the particular solution is thus

$$u = \frac{P_o}{k} \frac{1}{\sqrt{(1-\beta^2)^2 + (2\zeta\beta)^2}} \left\{ (1-\beta^2) \sin \Omega t - 2\zeta\beta \cos \Omega t \right\} \text{-----(3)}$$

- The complimentary solution (transient response) is:

$$u_{trans} = e^{-\zeta\omega t} (A \cos \omega_d t + B \sin \omega_d t) \text{-----(4)}$$

Damped Harmonic Vibration (Cont..)

- The complete solution is thus:

$$u = e^{-\zeta\omega t} \underbrace{(A \cos \omega_d t + B \sin \omega_d t)}_{\text{Transient}} + \underbrace{G_1 \sin \Omega t + G_2 \cos \Omega t}_{\text{Steady State}} \text{-----(5)}$$

where A and B are arbitrary constants determined from the initial conditions:

$$A = \frac{P_o}{k} \frac{2\zeta\beta}{(1-\beta^2)^2 + (2\zeta\beta)^2} + u_o, \quad B = \frac{P_o}{k} \frac{\omega}{\omega_d} \frac{2\beta\zeta^2 - \beta(1-\beta^2)}{(1-\beta^2)^2 + (2\zeta\beta)^2} + \frac{v_o + u_o\omega\zeta}{\omega_d} \text{-----(7)}$$

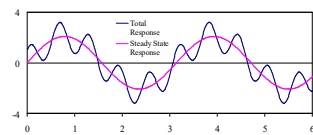
- The transient response is given by:

$$u_{trans} = e^{-\zeta\omega t} \left(u_o \cos \omega_d t + \frac{v_o + u_o\omega\zeta}{\omega_d} \sin \omega_d t \right) + \frac{P_o}{k} \frac{e^{-\zeta\omega t}}{(1-\beta^2)^2 + (2\zeta\beta)^2} \left[2\zeta\beta \cos \omega_d t + \frac{\omega}{\omega_d} [2\beta\zeta^2 - \beta(1-\beta^2)] \sin \omega_d t \right] \text{-----(8)}$$

Damped Harmonic Vibration (Cont..)

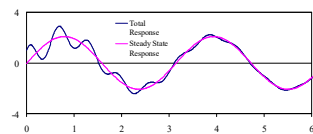
- Undamped Harmonic Vibration

- The transient response does not decay with time



- Damped Harmonic Vibration

- The transient response decay with time



Damped Harmonic Vibration (Cont..)

- The steady state response (Eq.3) of a damped harmonic vibration can be written in the alternative form as:

$$u = \rho \sin(\Omega t - \phi) \text{-----(9)}$$

- Where ρ is the amplitude of steady state response and ϕ is the phase angle by which the response lags the exciting force.

$$\rho = \frac{P_o}{k} \frac{1}{\sqrt{(1-\beta^2)^2 + (2\zeta\beta)^2}}, \quad \tan \phi = \frac{2\zeta\beta}{1-\beta^2}$$

Damped Harmonic Vibration : Magnification

- Dynamic load factor, i.e. the ratio of dynamic to static displacement, is given by:

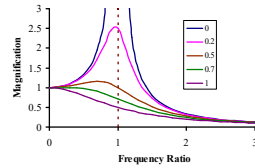
$$D(t) = \frac{1}{\sqrt{(1-\beta^2)^2 + (2\zeta\beta)^2}} \sin(\Omega t - \phi)$$

- The amplitude of dynamic load factor called the **dynamic magnification factor** is:

$$A_D = \frac{1}{\sqrt{(1-\beta^2)^2 + (2\zeta\beta)^2}}$$

- The plot between β and A_D shows that A_D is not maximum at $\beta=1.0$. The value of β to maximize A_D is obtained as:

$$\frac{dA_D}{d\beta} = 0 \Rightarrow \beta = \sqrt{1-2\zeta^2} \quad (A_D)_{\max} = \frac{1}{2\zeta\sqrt{1-\zeta^2}}$$



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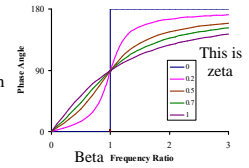
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Damped Harmonic Vibration : Phase Angle

- At resonance, i.e. $\beta=1.0$, all the curves pass through a single point ($\phi=90^\circ$).

$$\tan \phi = \frac{2\zeta\beta}{1-\beta^2}$$

- For $\beta < 1.0$ the phase angle is less than 90° .
- For $\beta > 1.0$ the phase angle is between 90° and 180° .
- For undamped system the steady state response is in phase ($\phi=0$) with the exciting force when $\beta < 1.0$ \rightarrow Zeta (0)
- For undamped system the steady state response is in out-of-phase ($\phi=180$) with the exciting force when $\beta > 1.0$.



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Resonance Response: Damped System

- At resonance ($\beta=1.0$) the steady state and transient responses are:

$$u_{steady} = -\frac{P_0}{2\zeta k} \cos \Omega t$$

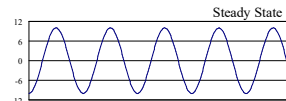
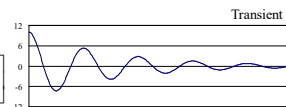
$$u_{trans} = \frac{P_0}{k} \frac{e^{-\zeta\omega_d t}}{2\zeta} \left[\cos \omega_d t + \frac{\omega_d \zeta}{\omega_d} \sin \omega_d t \right]$$

- In the initial phase the transient response will minimize the total response.

- When the transient response diminish, the system vibrates with the exciting force but lagging by 90° .

- For small damping, the total response is:

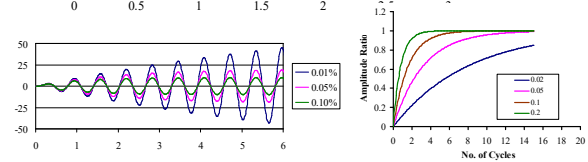
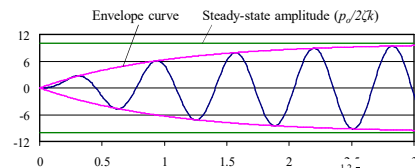
$$u_{total} = \frac{P_0}{k} \frac{1}{2\zeta} (e^{-\zeta\omega_d t} - 1) \cos \omega t$$



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Resonance Response: Damped System (Cont..)



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