Analysis of Statically Indeterminate Structures by the Displacement Method

Methods of structural analysis

The methods are classified into two groups:

- 1.Force method of analysis (Statics of building structures I)
	- **Primary unknowns are forces and compatibility of** displacements is written in terms of pre-selected redundant reactions and flexibility coefficients using force displacement relations.
	- **The unknown redundant reactions are evaluated** solving these equations.
	- The remaining reactions are obtained from equationsof equilibrium.

Methods of structural analysis

The methods are classified into two groups:

2. Displacement method of analysis

- **Primary unknowns are displacements.**
- **Equilibrium equations are written by expressing** the unknown joint displacements in terms of loads by using load-displacement relations.
- **Unknown joint displacements are calculated by** solving equilibrium equations.
- In the next step, the unknown reactions are computed from compatibility equations using force displacement relations.

Displacement method

- This method follows essentially the same steps for both statically determinate and indeterminate structures.
- $\mathcal{L}_{\mathcal{A}}$ Once the structural model is defined, the unknowns (joint rotations and translations) are automatically chosen unlike the force method of analysis (hence, this method is preferred to computer implementation).

Displacement method

Slope-Deflection Method

1.

- **IF In this method it is assumed that all deformations** are due to bending only. Deformations due to axial forces are neglected.
- 2. Direct Stiffness Method
	- **Deformations due to axial forces are not neglected.**

The Slope-deflection method was used for many years before the computer era. After the revolution occurred in the field of computing direct stiffnessmethod is preferred.

Slope-Deflection Method: Beams

Slope-Deflection Method: Beams

- Application of Slope-Deflection Equations to Statically <u>Indeterminate Beams:</u>
	- **The procedure is the same whether it is applied to** beams or frames.
	- It may be summarized as follows:
- 1. Identify all kinematic degrees of freedom for the given problem. Degrees of freedom are treated as unknownsin slope-deflection method.
- 2. Determine the fixed end moments at each end of the span to applied load (using table).
- 3. Express all internal end moments by slope-deflectionequations in terms of:
	- **Fixed end moments**
	- **near end and far end joint rotations**

Slope-Deflection Method: Beams

- 4. Write down one equilibrium equation for each unknown joint rotation. Write down as many equilibrium equations as there are unknown joint rotations. Solve the set of equilibrium equations for joint rotations.
- 5. Now substituting these joint rotations in the slopedeflection equations evaluate the end moments.
- 6. Evaluate shear forces and reactions.
- 7.Draw bending moment and shear force diagrams.

- 1. Degrees of freedom
	- **The continuous beam is kinematically indeterminate to** first degree. Only one joint rotation $\varphi_{\!b}$ is unknown. *b*

Slope-Deflection Method: Beams Example 1*F*= 20 kN

near end joint rotation

3. Express internal end moments by slope-deflection equations.

$$
M_{ab} = \frac{\overline{M}_{ab}}{l} + \frac{2\overline{EI}}{l} \left(2\overline{\varphi_a} + \overline{\varphi_b} \right)
$$
 for end joint rotation

fixed end momentflexural rigidity

$$
M_{ab} = \overline{M}_{ab} + \frac{2EI}{l_{ab}} (2 \cdot \varphi_a + \varphi_b) = -21 + \frac{2EI}{6} (2 \cdot 0 + \varphi_b) = -21 + \frac{EI}{3} \varphi_b
$$

$$
M_{ba} = \overline{M}_{ba} + \frac{2EI}{l_{ab}} (2 \cdot \varphi_b + \varphi_a) = 21 + \frac{2EI}{6} (2 \cdot \varphi_b + 0) = 21 + \frac{2 \cdot EI}{3} \varphi_b
$$

$$
M_{ba} = \overline{M}_{ba} + \frac{2EI}{l_{ab}} (2 \cdot \varphi_b + \varphi_a) = 21 + \frac{2EI}{6} (2 \cdot \varphi_b + 0) = 21 + \frac{2 \cdot EI}{3} \varphi_b
$$

$$
M_{bc} = \overline{M}_{bc} + \frac{2EI}{l_{bc}} (2 \cdot \varphi_b + \varphi_c) = -5,33 + \frac{2EI}{4} (2 \cdot \varphi_b + 0) = -5,33 + EI \cdot \varphi_b
$$

$$
M_{cb} = \overline{M}_{cb} + \frac{2EI}{l} (2 \cdot \varphi_c + \varphi_b) = 5,33 + \frac{2EI}{4} (2 \cdot 0 + \varphi_b) = 5,33 + \frac{EI}{2} \cdot \varphi_b
$$

$$
M_{cb} = \overline{M}_{cb} + \frac{2EI}{l_{bc}} (2 \cdot \varphi_c + \varphi_b) = 5,33 + \frac{2EI}{4} (2 \cdot 0 + \varphi_b) = 5,33 + \frac{EI}{2} \cdot \varphi_b
$$

Slope-Deflection Method: Beams Example 1*a* $q_1 = 2$ kN/m 33 $\overrightarrow{3}$ 4 *bb* c $q_2 = 4$ kN/m $F = 20$ kN

- 4. Equilibrium equations (write one equilibrium equation for each unknown joint rotation)
- \mathbb{R}^n ■ End moments are expressed in terms of unknown rotation φ_b .
Now, the required equation to solve for the rotation φ_b is the n φ_b is the uilibrium equations (write one equilib
ch unknown joint rotation)
End moments are expressed in terms of un
Now, the required equation to solve for the
moment equilibrium equation at support *b*.

$$
\sum M_b = 0: \quad M_{ba} + M_{bc} = 0
$$

21 + $\frac{2EI}{3}$ · φ_b – 5,33 + EI · φ_b = 0
1,667 · EI · φ_b + 15,667 = 0 $\Rightarrow \varphi_b$ = $-\frac{15,667}{1,667 \cdot EI} = -\frac{9,4}{EI}$

Slope-Deflection Method: Beams Example 1*a* $q_1 = 2$ kN/m 33 $\overrightarrow{3}$ 4 *bb* c $q_2 = 4$ kN/m *F*= 20 kN

5. End moments

 $\overline{}$ **A** After evaluating φ_b , substitute it to evaluate beam end moments.

$$
M_{ab} = -21 + \frac{EI}{3} \varphi_b = -21 + \frac{EI}{3} \cdot \left(-\frac{9.4}{EI} \right) = -24.13 \text{ kNm}
$$

\n
$$
M_{ba} = 21 + \frac{2 \cdot EI}{3} \varphi_b = 21 + \frac{2 \cdot EI}{3} \cdot \left(-\frac{9.4}{EI} \right) = 14.73 \text{ kNm}
$$

\n
$$
M_{bc} = -5.33 + EI \cdot \varphi_b = -5.33 + EI \cdot \left(-\frac{9.4}{EI} \right) = -14.73 \text{ kNm}
$$

\n
$$
M_{cb} = 5.33 + \frac{EI}{2} \cdot \varphi_b = 5.33 + \frac{EI}{2} \cdot \left(-\frac{9.4}{EI} \right) = 0.63 \text{ kNm}
$$

6. Shear forces and reactions

Service Service Now, reactions at supports are evaluated using equilibrium equations. Shear forces are equal to plus/minus this reactions.

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$$
R_{az} = R_{ab} = 17,57 \text{ kN}
$$

\n
$$
R_{bz} = R_{ba} + R_{bc} = 14,43 + 11,53 = 25,96 \text{ kN}
$$

\n
$$
R_{cz} = R_{cb} = 4,47 \text{ kN}
$$

7.. Draw shear force and bending moment diagrams.

7.Draw shear force and **bending moment** diagrams.

- 1. Degrees of freedom
	- **The continuous beam is kinematically indeterminate to** second degree.
		- > The 1st possibility of solution two unknown joint rotation φ_b , φ_c ($\pmb{\varphi}_a$ = 0) - two required equations to solve for the rotation $\pmb{\varphi}_b$, $\pmb{\varphi}_c$ are the moment equilibrium equations at support *b* and *^c*.

2. Fixed end moments are calculated referring to the table.

$$
\overline{M}_{ab} = -\frac{1}{12} \cdot q \cdot l_{ab}^2 = -\frac{1}{12} \cdot 3 \cdot 8^2 = -16 \text{ kNm}
$$

$$
\overline{M}_{ba} = +\frac{1}{12} \cdot q \cdot l_{ab}^2 = 16 \text{ kNm}
$$

$$
\overline{M}_{bc} = -\frac{1}{8} \cdot F \cdot l_{bc} = -\frac{1}{8} \cdot F \cdot l_{bc} = -\frac{1}{8} \cdot 10 \cdot 6 = -7,5 \text{ kNm}
$$

$$
\overline{M}_{cb} = +\frac{1}{8} \cdot F \cdot l_{bc} = +\frac{1}{8} \cdot F \cdot l_{bc} = +\frac{1}{8} \cdot 10 \cdot 6 = 7,5 \text{ kNm}
$$

Slope-Deflection Method: Beams Example 2 *aq*= 3 kN/m8 $8 \rightarrow 3$ *b* $b \underline{\triangle}$ c $F_1 = 10 \text{ kN}$ 3 $\overrightarrow{3}$ $\overrightarrow{3}$ $F_2 = 5$ kN $EI_{ab} = EI$ \bar{d} $EI_{bc} = 2EI$

3. Express internal end moments by slope-deflection equations.

$$
M_{ab} = \overline{M}_{ab} + \frac{2EI_{ab}}{l_{ab}} (2 \cdot \varphi_a + \varphi_b) = -16 + \frac{2EI}{8} (2 \cdot 0 + \varphi_b) = -16 + \frac{EI}{4} \varphi_b
$$

\n
$$
M_{ba} = \overline{M}_{ba} + \frac{2EI_{ab}}{l_{ab}} (2 \cdot \varphi_b + \varphi_a) = 16 + \frac{2EI}{8} (2 \cdot \varphi_b + 0) = 16 + \frac{EI}{2} \varphi_b
$$

\n
$$
M_{bc} = \overline{M}_{bc} + \frac{2EI_{bc}}{l_{bc}} (2 \cdot \varphi_b + \varphi_c) = -7,5 + \frac{2 \cdot 2EI}{6} (2 \cdot \varphi_b + \varphi_c) = -7,5 + \frac{4}{3} EI \cdot \varphi_b + \frac{2}{3} EI \cdot \varphi_c
$$

\n
$$
M_{cb} = \overline{M}_{cb} + \frac{2EI_{bc}}{l_{bc}} (2 \cdot \varphi_c + \varphi_b) = +7,5 + \frac{2 \cdot 2EI}{6} (2 \cdot \varphi_c + \varphi_b) = 7,5 + \frac{4}{3} EI \cdot \varphi_c + \frac{2}{3} EI \cdot \varphi_b
$$

\n
$$
M_{cd} = -F_2 \cdot 3 = -15 \text{ kNm}
$$

 $=-F_2 \cdot 3 = -15$ kNm

Slope-Deflection Method: Beams Example 2 *aq*= 3 kN/m8 $8 \rightarrow 3$ *b* $b \underline{\triangle}$ c $F_1 = 10 \text{ kN}$ 3 $\overrightarrow{3}$ $\overrightarrow{3}$ $F_2 = 5$ kN

- 4. Equilibrium equations (write two equilibrium equations for two unknown joint rotations)
	- **End moments are expressed in terms of unknown rotations. Now, the** required equations to solve for the rotations are the moment equilibrium equations at supports *^b*and *^c*.

nd *c*.
\n
$$
\sum M_b = 0: M_{ba} + M_{bc} = 0
$$
\n
$$
\sum M_c = 0: M_{cb} + M_{cd} = 0
$$
\n
$$
16 + \frac{EI}{2}\varphi_b - 7.5 + \frac{4}{3}EI \cdot \varphi_b + \frac{2}{3}EI \cdot \varphi_c = 0
$$
\n
$$
\frac{7.5 + \frac{4}{3}EI \cdot \varphi_c + \frac{2}{3}EI \cdot \varphi_b - 15 = 0}{3}
$$

$$
\varphi_b = -\frac{8,167}{EI} \quad \varphi_c = \frac{9,708}{EI}
$$

d

5. End moments

 \Box After evaluating φ_b , φ_c , substitute them to evaluate beam end
moments moments.

$$
M_{ab} = -16 + \frac{EI}{4}\varphi_b = -16 + \frac{EI}{4}\left(-\frac{8,167}{EI}\right) = -18,04 \text{ kNm}
$$

$$
M_{ba} = 16 + \frac{EI}{2}\varphi_b = 16 + \frac{EI}{2}\left(-\frac{8,167}{EI}\right) = 11,92 \text{ kNm}
$$

$$
M_{ab} = -16 + \frac{EI}{4}\varphi_b = -16 + \frac{EI}{4}\left(-\frac{8,167}{EI}\right) = -18,04
$$

$$
M_{ba} = 16 + \frac{EI}{2}\varphi_b = 16 + \frac{EI}{2}\left(-\frac{8,167}{EI}\right) = 11,92
$$
 kNm

$$
M_{bc} = -7.5 + \frac{4}{3}EI \cdot \varphi_b + \frac{2}{3}EI \cdot \varphi_c = -7.5 + \frac{4}{3}EI \cdot \left(-\frac{8.167}{EI}\right) + \frac{2}{3}EI \cdot \frac{9.708}{EI} = -11.92 \text{ kNm}
$$

$$
M_{ba} = 16 + \frac{24}{2}\varphi_b = 16 + \frac{24}{2}\left(-\frac{340}{EI}\right) = 11,92 \text{ kNm}
$$

\n
$$
M_{bc} = -7,5 + \frac{4}{3}EI \cdot \varphi_b + \frac{2}{3}EI \cdot \varphi_c = -7,5 + \frac{4}{3}EI \cdot \left(-\frac{8,167}{EI}\right) + \frac{2}{3}EI \cdot \frac{9,708}{EI} = -11,9
$$

\n
$$
M_{cb} = 7,5 + \frac{4}{3}EI \cdot \varphi_c + \frac{2}{3}EI \cdot \varphi_b = 7,5 + \frac{4}{3}EI \cdot \frac{9,708}{EI} + \frac{2}{3}EI \cdot \left(-\frac{8,167}{EI}\right) = 15 \text{ kNm}
$$

\n
$$
M_{cd} = -F_2 \cdot 3 = -15 \text{ kNm}
$$

 $=-F_2 \cdot 3 = -15$ kNm

6. <u>Shear forces a</u>nd reactions.

$$
V_{ab} = R_{ab} = \frac{1}{l_{ab}} \left(q_1 \cdot \frac{l_{ab}^2}{2} - M_{ab} - M_{ba} \right)
$$

$$
V_{ab} = \frac{1}{8} \left(3 \cdot \frac{8^2}{2} + 18,04 - 11,92 \right) = 12,771
$$

$$
V_{ab} = \frac{1}{8} \left(3 \cdot \frac{8^2}{2} + 18,04 - 11,92 \right) = 12,77 \text{ kN}
$$

$$
V_{ab} = 8\left(\frac{3}{2} - \frac{1}{1000} + \frac{l_{ab}^2}{2}\right) = 12,77 \text{ m/s}
$$

$$
V_{ba} = -R_{ba} = -\frac{1}{l_{ab}} \left(q_1 \cdot \frac{l_{ab}^2}{2} + M_{ab} + M_{ba}\right)
$$

$$
V_{ba} = -\frac{1}{8} \left(3 \cdot \frac{8^2}{2} - 1804 + 1192\right) = -1123 \text{ m}
$$

$$
V_{ba} = -\frac{1}{8} \left(3 \cdot \frac{8^2}{2} - 18,04 + 11,92 \right) = -11,23 \text{ kN}
$$

6. Shear forces and <u>reactions.</u>

$$
R_{az} = R_{ab} = 12,77 \text{kN}
$$

\n
$$
R_{bz} = R_{ba} + R_{bc} = 11,23 + 4,49 = 15,72 \text{kN}
$$

\n
$$
R_{cz} = R_{cb} + F_2 = 5,51 + 5 = 10,51 \text{kN}
$$

7.Draw shear force and bending moment diagrams.

- 1. Degrees of freedom
	- **The continuous beam is kinematically indeterminate to** second degree.
		- > The 2nd possibility of solution solve only one unknown joint rotation φ_b (φ_a = 0, joint rotation φ_c is not necessary to solution because the moment in the cantilever portion M_c is known \Rightarrow beam portion *bc* is taken as fixed - hinged).

2. Fixed end moments are calculated referring to the table.

$$
\frac{8}{\sqrt{h}}
$$
\n
$$
3 + 3 + 3 + \frac{1}{2} \sqrt{h}
$$
\n
$$
\frac{1}{\sqrt{h}}
$$
\n
$$
k = \frac{1}{12} \cdot q \cdot l_{ab}^2 = -\frac{1}{12} \cdot 3 \cdot 8^2 = -16 \text{ kNm}
$$
\n
$$
M_{ba} = +\frac{1}{12} \cdot q \cdot l_{ab}^2 = 16 \text{ kNm}
$$
\n
$$
M_{bc} = -\frac{3}{16} \cdot F \cdot l_{bc} + M_c \cdot \frac{l_{bc}^2 - 3 \cdot b^2}{2 \cdot l_{bc}^2} = -\frac{3}{16} \cdot F \cdot l_{bc} + \frac{M_c}{2 \cdot l_{bc}^2} = -\frac{3}{16} \cdot F \cdot l_{bc} + \frac{M_c}{2 \cdot l_{bc}^2} = -\frac{3}{16} \cdot F \cdot l_{bc} + \frac{M_c}{2 \cdot l_{bc}^2} = -\frac{3}{16} \cdot F \cdot l_{bc} + \frac{M_c}{2 \cdot l_{bc}^2} = -\frac{3}{16} \cdot 6 + \frac{M_c}{2} = -\frac{3}{16} \cdot 10 \cdot 6 + \frac{15}{2} = -3.75 \text{ kNm}
$$

3. Express internal end moments by slope-deflection equations.

$$
M_{ab} = \overline{M}_{ab} + \frac{2EI}{l} (2 \cdot \varphi_a + \varphi_b)
$$

\n
$$
M_{ab} = \overline{M}_{ab} + \frac{3EI}{l} \cdot \varphi_a
$$

\n
$$
M_{ab} = \overline{M}_{ab} + \frac{2EI_{ab}}{l_{ab}} (2 \cdot \varphi_a + \varphi_b) = -16 + \frac{2EI}{8} (2 \cdot 0 + \varphi_b) = -16 + \frac{EI}{4} \varphi_b
$$

\n
$$
M_{ba} = \overline{M}_{ba} + \frac{2EI_{ab}}{l_{ab}} (2 \cdot \varphi_b + \varphi_a) = 16 + \frac{2EI}{8} (2 \cdot \varphi_b + 0) = 16 + \frac{EI}{2} \varphi_b
$$

\n
$$
M_{bc} = \overline{M}_{bc} + \frac{3EI_{bc}}{l_{bc}} \cdot \varphi_b = -3.75 + \frac{3 \cdot 2EI}{6} \cdot \varphi_b = -3.75 + EI \cdot \varphi_b
$$

 $b = 3, 73$
b $\varphi_b = 3, 73$
b φ_b

 $6 \t\t 6$

bc

l

- 4. Equilibrium equations (write one equilibrium equation for each unknown joint rotation)
- $\overline{}$ ■ End moments are expressed in terms of unknown rotation φ_b .
Now, the required equation to solve for the rotation φ_b is the n $\varphi_{\!b}$ is the moment equilibrium equation at support *b*.8 \longrightarrow 3 \longrightarrow 3

one equilibrium equ

cone equilibrium equ

cone for the rotation γ

at support *b*.

$$
\sum M_b = 0: \quad M_{ba} + M_{bc} = 0
$$

16 + $\frac{EI}{2}\varphi_b - 3{,}75 + EI \cdot \varphi_b = 0$
1,5 · $EI \cdot \varphi_b + 12{,}25 = 0 \Rightarrow \varphi_b = -\frac{12{,}25}{1{,}5 \cdot EI} = -\frac{8{,}167}{EI}$

5. End moments

Service Service ■ After evaluating φ_{b} , substitute it to evaluate beam end moments.

$$
M_{ab} = -16 + \frac{EI}{4} \cdot \varphi_b = -16 + \frac{EI}{4} \cdot \left(-\frac{8,167}{EI}\right) = -18,04 \text{ kNm}
$$

$$
M_{ba} = 16 + \frac{EI}{2} \cdot \varphi_b = 16 + \frac{EI}{2} \cdot \left(-\frac{8,167}{EI}\right) = 11,92 \text{ kNm}
$$

$$
M_{bc} = -3,75 + EI \cdot \varphi_b = -3,75 + EI \cdot \left(-\frac{8,167}{EI}\right) = -11,92 \text{ kNm}
$$

Then the procedure is the same as for the 1st possibility of solution.